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Influence of the Extrusion Cavity Shape on the Ultimate Extrusion Force Value During Pellet Extrusion through Multiple-Cavity Dies

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ABSTRACT

The article describes the results of experimental research that allowed us to compare the influence of the forming channel shape and its length on the limit value of the force occurring in the process of extruding crystallized carbon dioxide using multi-channel dies. To carry out the research, the authors adopted a method described in the literature, which allows for experimenting using a sublimating material under normal conditions, at a temperature of -78.4°C. Analysis of variance was used for the statistical analysis of the experiment results, which allowed to determine the level of statistical significance of differences between individual populations. Additionally, based on the median and regression values, an attempt was made to fit a function describing the change in the tested values as a function of geometric parameters. This allowed for comparison of the impact of individual variables. The authors did not observe the research results in the available literature, they would discuss the influence of the channel shape of multi-channel systems on the value of the confining force. Therefore, according to the authors, the presented results constitute an important supplement to the available knowledge.

Keywords: extrusion, compaction, dry ice, cavity shape, multichannel die.

INTRODUCTION

Transporting materials plays a vital part in the growth of civilisations. For centuries people have been looking for methods allowing to shorten the haulage time or increase the haulage distance. In the case of temperature-controlled goods, haulage technology developments must be accompanied by improvement of the associated storage methods. These temperature-controlled goods in transit include foodstuffs and medical products, in particular pharmaceuticals such as SARS CO-VID-19 vaccines with -80°C to -60°C [1] longterm storage temperature requirement. Dry ice shipping containers are one of the solutions to obtain these storage temperatures. Dry ice (DI) is solid carbon dioxide, in normal ambient conditions having a temperature of -78.4 °C [2]. As we see in the phase change diagram (Fig. 1), dry

Therefore, dry ice is generally subjected to mechanical processing, with the aim to improve its use efficiency. On industrial scale, special DI ram extruders are used for this purpose. The

DI ram extruders are used for this purpose. The working system schematic is presented in Figure 2. The system of these machines has been extensively described by Górecki et al. 2019 [5]. Dry ice extrusion process consists of four subsequent stages. In the first stage, liquid carbon dioxide is injected into the compression chamber. This triggers an adiabatic process decreasing the material's temperature and changing its state from liquid to solid. In the next stage, the ram (2) moves along the compression chamber axis, thus compressing the initially loose material. This stage ends when the value of F_c has reached the value

ice will sublimate in the said conditions. As it has

been reported in earlier studies, sublimation ad-

versely affects the cooling process efficiency [3].



Figure 2. Extruder working system including multiple-cavity die, 1 - compression chamber, 2 - ram, 3 - multiple cavity die, 4 - loose dry ice, $F_R - \text{resisting force}$, $F_C - \text{compacting force}$ [5]

of extrusion resistance force F_{R} of the die in question, multiple-cavity type in this case. In the next stage compressed DI is pushed through the die cavity formed on the way to the pre-defined end-product shape, such as 3 mm pellets. As soon as the ram has reached the end-of-travel position, it is withdrawn to the home position and the cycle begins anew. The process of dry ice production been extensively described in the literature and the interested readers are advised to consult the referenced articles [6] for further details.

As mentioned, the purpose of this process is to reduce the material's surface area to reduce its sublimation rate. For this reason, the density of DI pellets ρ_{DI} [7] has been chosen as the extrudate quality indicator. Its value is directly related to the geometric features of multiple-cavity die (Fig. 3) and the billet length l_T before it gets split and extruded in the third stage of the process. In our earlier articles [8,9] we described the relationship between F_{R} and the geometric features of conical convergent multiple-cavity dies. The relevant geometric features of the analysed dies are shown in Figure 3. Wałęsa et al. (2022) presented the outcomes of a simulation study investigating the performance of single-cavity dies featuring four different convergent section geometries: conical, concave, convex and concave-concave, as shown in Figure 4. In the case of a constant value of the inlet diameter (D) and outlet diameter (d), the shape of the conic section determines the length of this section (h) or the convergence angle (α). For a spherical, convex, or concave convergent section, specify the radius value $(R_1 \text{ or } R_2)$ and the section's length $(h_1 \text{ or } h_2)$. In the case of the fourth type of channel, fixed from the connection of the two previous ones, it is necessary to determine the parameters for both sections independently or to ascertain the total length of the convergent section and define the proportion of each share, e.g.,



Figure 3. Conical convergent multiple-cavity die, (a) overview, (b) single cavity detail, d – die land diameter, α_R – side inclination of the conical section, l_C – conical section length, l_L – die land length [9]



Figure 4. Shapes of the dies under analysis: (a) conical-cylindrical (CC), (b) with a convex spherical compression section, (c) with a concave spherical compression section (CS), (d) with a convex section followed by concave section (CCS) [13]

1 to 1. The convex spherical compression section was left out based on the results of earlier studies that revealed problems with obtaining laminar flow of the material during extrusion and maintaining cohesion. On the other hand, simulations with dies including a concave spherical compression section and a convex section followed by concave section showed that dies including a non-conical convergent section may be an option. This shows that the aforesaid shapes should be taken into consideration in research and development work on improving the efficiency of DI extrusion process using single-cavity dies.

The literature on the subject of multiple-cavity dies apparently does not include any reports on the effect of convergent sections shapes other than conical. In turn, the results presented in this article concern dies including convex and convex followed by concave convergent sections. Therefore, the article is intended to fill the revealed gap in knowledge. In addition, the said results may serve as reference data in DI extrusion numerical simulations using artificial intelligence techniques that are arousing great interest among machine design engineers of various fields [10-12].

METHODOLOGY AND MATERIALS

Materials

The material used in the experiments was loose dry ice of ca. 550 kg/m³ bulk density [14]. It is obtained by expanding liquid carbon dioxide stored at -18 Celsius degrees from 20 bar to atmospheric pressure. The particle size of loose dry ice obtained in this way was determined by Liu et. al. 2010. It ranged 20–100 μ m, depending on the method of expansion [15]. Figure 5 shows dry ice in different forms: loose (first from the left) and compacted into 3 mm pellets (second from the left) and 16 mm initial cross-section pellets (first from the right). The apparent non-cylindricity of the pellets results from the loss of material due to sublimation.

The dies

Multiple-cavity dies including CC, CS, and CCS convergent sections were used in the experiments. In each die type five different samples were prepared, featuring different convergent section lengths, ranging 1 mm to 9 mm. The geometric features of the analysed dies are shown in Figure 6, and the values are given in Table 1. For comparison purposes, the inlet diameter D, outlet diameter d, overall die length L and extrusion cavity layout remained the same between the tested dies. The experiments caused evident distortion of some cavities, and hence some h_c values were not included in Table 1.

Compression force measurement method

This study used the method described by Biszczanik et al. 2023, originally used to investigate the effect of geometric features of single-cavity dies on the DI extrusion process

 Table 1. Geometric features of the analysed multiplecavity dies

| Die type | | | <i>h_c</i> [mm] | | |
|----------|---|---|---------------------------|---|---|
| CC | 1 | 3 | 5 | 7 | 9 |
| CS | 1 | 3 | 5 | 7 | 9 |
| CCS | 1 | 3 | 5 | 7 | 9 |



Figure 5. Dry ice, from left to right: loose, 3 mm pellets, 16 mm pellets [16]











Figure 6. Multiple-cavity dies featuring the following convergent section types: (a) CC, (b) CS, (c) CCS, D equal to 5 mm, d equal to 3 mm, L equal to 18 mm



Figure 7. Dry ice compression and extrusion test facility, 1 – MTS Insight 50 kN test frame,
2 – Upper sleeve with the compaction chamber contained inside it, 3 – Lower sleeve, 4 – Guide assembly, 5 – Ram, 6 – Multiple-cavity die, 7 – Loose dry ice, 8 – Dry ice pellets

characteristics [16]. The experimental set-up is presented in Figure 7. This set-up allowed measurement of the applied force during the loose dry ice (Fig. 7, label 7) compression and extrusion stages, the extrudate being 3 mm pellets (Fig. 7, label 6), as defined by the parameter d of the applied multiple-cavity dies. In order to allow fitting of multiple-cavity dies (Fig. 7, label 6), the test set-up described by Biszczanik was modified to accommodate the greater compression chamber (Fig. 7, label 2) diameter of 38 mm and the associated increase in the ram diameter to 37.8mm (Fig. 7 label 3). The test facility was composed of MTS Insight 50 kN test frame (MTS Systems, USA, Eden Prairie) with the above-described test set-up mounted in the test frame grips. Dedicated TestWork 4.0 software by MTS Systems (USA, Eden Prairie) was used to control the test facility and log the measurement results with 50 Hz frequency. The linear motion of the top jaw pushes the top plate with the ram mounted thereon (Fig. 7, label 5) down at a speed of 5 mm/sec. This motion reduced the available space inside the compression chamber, resulting in compression of the material



Figure 8. The insulated dry ice storage container and the condition the experimental set-up components, 1 -insulated container, 2 -dry ice, 3 -experimental set-up [17]

contained therein (Fig. 7, label 7). The ram motion continued until it had reached the bottom end position located 1 mm above the top surface of the die. Next the ram was withdrawn to its home position and the next test cycle could begin. In order to reduce the effect of sublimation on the test results, all the parts that would get in contact with dry ice were pre-cooled. For this purpose, the sleeve assembly including the ram was placed in the DrIcy 30 firmy insulated dry ice pellet storage container manufactured by Melform of Monasterolo di Savigliano, Italy. The container containing the test set-up is shown in Figure 8. Each test series comprised three repetitions and the test set-up components were held in the insulated container for 30 minutes before the first series and then for 10 minutes in between.

RESULTS AND DISCUSSION

The following diagrams represents the change of compression force F_c as a function of ram displacement s obtained for CC die of h=5. The parameters measured to determine the effect of the geometric features for different convergent section types were the ultimate force F_c^{max} and the distance between the ram face and the multiple-cavity die $s(F_c^{max})$. The readings were logged as populations, in order to allow the use of the analysis of variance method (ANOVA) and determine the statistical significance value p of the hypotheses that the value of F_c^{max} had a bearing on:

 convergent section shape, irrespective of the section length (5 tests for h=const; type=var),



Figure 9. Change of F_c as a function of *s*: F_c^{max} – limit value of $F_c(s)$ during a trial

- convergent section length, irrespective of the section length (3 tests for type=const; h=var),
- convergent section surface area (1 test for S=var).

One-way analysis of variance along with Tukey post hoc test were used to assess the mean differences of F_c^{max} and ρ_{CCD} between the data groups. The analysis of data was performed using the Statistica software (Version 13.3, TIBCO Software Inc.). Statistical significance defined

Table 2. Values of p between the compared populationsfor convergent section of CC shape

| CC | h=3 | h=5 | h=7 |
|-----|----------|----------|----------|
| h=3 | | 0.000129 | 0.000129 |
| h=5 | 0.000129 | | 0.000129 |
| h=7 | 0.000129 | 0.000129 | |

Table 3. Values of p between the compared populations for convergent section of CS shape

| CS | h=3 | h=5 | h=7 |
|-----|----------|----------|----------|
| h=3 | | 0.003222 | 0.000130 |
| h=5 | 0.003222 | | 0.014358 |
| h=7 | 0.000130 | 0.014358 | |

Table 4. Values of *p* between the compared populations for convergent section of CCS shape

| CCS | h=3 | h=5 | h=7 |
|-----|----------|----------|----------|
| h=3 | | 0.000127 | 0.000127 |
| h=5 | 0.000127 | | 0.000127 |
| h=7 | 0.000127 | 0.000127 | |

Table 5. Values of p between the compared populations for convergent section of h=3 mm

| h=3 | CC | CS | CCS |
|-----|----------|----------|----------|
| CC | | 0.000129 | 0.003899 |
| CS | 0.000129 | | 0.000129 |
| CCS | 0.003899 | 0.000129 | |

Table 6. Values of p between the compared populations for convergent section of h=5 mm

| h=5 | CC | CS | CCS |
|-----|----------|----------|----------|
| CC | | 0.000134 | 0.000129 |
| CS | 0.000134 | | 0.000129 |
| CCS | 0.000129 | 0.000129 | |

| h=7 | CC | CS | CCS |
|-----|----------|----------|----------|
| CC | | 0.043723 | 0.000129 |
| CS | 0.043723 | | 0.000129 |
| CCS | 0.000129 | 0.000129 | |

Table 7. Values of p between the compared populations for convergent section of h = 7 mm

Table 8. Key descriptive statistics for verification of the effect of h on the value of F for the CC cavity shape

| сс | AVR [kN] | Min. [kN] | Max. [kN] | Q25 [kN] | Q50 [kN] | Q75 [kN] |
|-----|-------------|--------------|--------------|-------------|-------------|-------------|
| h=3 | 17. 11 | 16.04 | 18. 29 | 16.33 | 17.18 | 17.64 |
| h=5 | 22.62 | 19.15 | 24.11 | 22.12 | 22.93 | 23.97 |
| h=7 | 33.00 | 29.17 | 34.72 | 33. 77 | 34.05 | 34.50 |

Table 9. Key descriptive statistics for verification of the effect of h on the value of F for the CS cavity shape

| cs | AVR [N] | Min. [N] | Max. [N] | Q25 [N] | Q50 [N] | Q75 [N] |
|-----|---------|-------------|-------------|------------|------------|------------|
| h=3 | 23.35 | 21.65 | 29.84 | 22.18 | 22.44 | 22.63 |
| h=5 | 27.67 | 25.31 | 31.88 | 26.34 | 27.12 | 29.47 |
| h=7 | 31.07 | 27.67 | 36.06 | 27.82 | 30.80 | 33.08 |

Table 10. Key descriptive statistics for verification of the effect of h on the value of F for the CCS cavity shape

| | | | | | | J 1 |
|-----|-------------|--------------|--------------|-------------|-------------|-------------|
| ccs | AVR [kN] | Min. [kN] | Max. [kN] | Q25 [kN] | Q50 [kN] | Q75 [kN] |
| h=3 | 15.87 | 14.36 | 16.75 | 15.61 | 15.84 | 16.54 |
| h=5 | 7.00 | 5.25 | 9.06 | 5.32 | 7.38 | 8.39 |
| h=7 | 24.37 | 21.04 | 28.20 | 23.70 | 23.89 | 25.69 |

Table 11. Key descriptive statistics for verification of the effect of the convergent section shape on the value of F for h = 3 mm

| h=3 | AVR [kN] | Min. [kN] | Max. [kN] | Q25 [kN] | Q50 [kN] | Q75 [N] |
|-----|-------------|--------------|--------------|-------------|-------------|------------|
| СС | 17.11 | 16.04 | 18.29 | 16.33 | 17.18 | 17.64 |
| CS | 22.54 | 21.66 | 24.14 | 22.18 | 22.44 | 22.63 |
| CCS | 15.87 | 14.36 | 16.75 | 15.61 | 15.84 | 16.54 |

Table 12. Key descriptive statistics for verification of the effect of the convergent section shape on the value of F for h = 5 mm

| h=5 | AVR [kN] | Min. [kN] | Max. [kN] | Q25 [kN] | Q50 [kN] | Q75 [kN] |
|-----|-------------|--------------|--------------|-------------|-------------|-------------|
| СС | 22.62 | 19.15 | 24.11 | 22.12 | 22.93 | 23.97 |
| CS | 27.67 | 25.31 | 31.88 | 26.34 | 27.12 | 29.47 |
| CCS | 7.06 | 5.25 | 9.06 | 5.32 | 7.38 | 8.39 |

Table 13. Key descriptive statistics for verification of the effect of the convergent section shape on the value of F for h = 7 mm

| h=7 | AVR [kN] | Min. [kN] | Max. [kN] | Q25 [kN] | Q50 [kN] | Q75 [kN] |
|-----|-------------|--------------|--------------|-------------|-------------|-------------|
| СС | 33.99 | 32.83 | 34. 72 | 33.77 | 34.05 | 34.50 |
| CS | 31.07 | 27.67 | 36.06 | 27.82 | 30.80 | 33.08 |
| CCS | 24.37 | 21.04 | 28.20 | 23.70 | 23.89 | 25.69 |

by the value of p below 0.05 was applied in all the performed one-way comparisons. The null hypotheses were verified before ANOVA. First Shapiro-Wilk test was applied to check normality of the data in the populations. The value of p varied between 0.0593 and 0.7401. With this value, all the test samples were found to have Gaussian distribution. In the next step, the equality of variances was checked between the compared populations. The statistical significance p ranged 0.0501 to 0.8055 thus rejecting the null hypothesis of homogeneity of variance in the respective samples. This allowed verification of the statistical significance value with the use of the respective samples and their populations. The values of p were smaller than 0.01 for the respective groups, this confirming statistical significance of differences between the populations. The p-value details for the respective populations are given in the Tables 2-7 and the key descriptive statistics are given in Tables 8-13.

The results given in the tables indicate statistically significant differences between the compared samples. Next regression analysis was performed using CurveExpert Professional (version 2.7.3, by Daniel G. Hyams) for the purpose of fitting the function in order to describe the effect of the convergent section length and type of the F_{C}^{max} value. With the convergent section type being the variable, the total inside surface area of the section was determined for regression purposes. To this end, the nominal variables CC, CS and CCS were converted to ratio variables per mm². In most of the derived functions, linear function was used as the reference value and another non-linear function was used to obtain higher R correlation and R² goodness-of-fit values. The derived functions accompanied by parameters are given in Table 14 and represented by diagrams in Figure 10.

The obtained coefficient of the determination (R^2) values show that Hoerl function yielded high goodness-of-it level in most cases. In the case of



Figure 10. Regression curves for the respective data samples (a). f(h) for CC shaped cavity;
(b) f(h) for CS shaped cavity; (c) f(h) for CCS shaped cavity, (d) f(A) for convergent sections of h=3; (e) f(A) for convergent sections of h=5; (f) f(A) for convergent sections of h=7

the sample used to estimate the change of F_C^{max} as a function of the h = 5 mm convergent section surface area the program could not determine the function factors, and hence regressions were carried out using a fourth degree polynomial. For the samples used for estimating the effect of h on the value of F_C^{max} we see that h has a linear effect on F_C^{max} in the case of CC and CS shaped cavities. Such linear relationship was not found for CCS of h = 5 data.

The same applies to the effect of the convergent section surface area on the value of F_{C}^{max} .

| Parameter | Description | Function | R | R ² |
|---------------------------------------|-------------|--|--------|----------------|
| F _c ^{max} (CC,h) | Linear | 4.38 [.] 10 ³ +3.973 [.] 10 ^{3.} h | 0.9615 | 0.9245 |
| | Hoerl | 1.35 [.] 10 ^{4.} 1.329h [.] h ^{.5.67} | 0.9765 | 0.9536 |
| F _c ^{max} (CS,h) | Linear | 1.4 [.] 10 ⁴ +2.51 [.] 10 ^{3.} h | 0.6557 | 0.4300 |
| | Hoerl | 1.03 [.] 10 ^{4.} 0.91 ^{h.} h ^{0.91} | 0.6670 | 0.4450 |
| F _c ^{max} (CCS,h) | Linear | 5.13 [.] 10 ³ +2.12 [.] 10 ^{3.} h | 0.4797 | 0.2301 |
| | Hoerl | 2.78 [.] 10 ^{6.} 13.69 ^{h.} h ^{-11.85} | 0.9807 | 0.9617 |
| F _c ^{max} (h=3,A) | Linear | -3.99 [.] 10 ³ +15.91 [.] h | 0.2723 | 0.0741 |
| | Sinusoidal | -3.04 [.] 10 ³ +2.76 [.] 10 ^{3.} cos(8.29 [.] 10 ^{.3.} h+1.22) | 0.4551 | 0.2071 |
| F _c ^{max} (h=5,A) | Linear | -1.07 [.] 10⁵+56.7 [.] h | 0.8907 | 0.7934 |
| | Polynomial | -6.9·10⁵+1.92·10²·h+0.13·h²+2.2·10⁻⁵·h³ -2.47·10⁻8·h⁴ | 0.9833 | 0.9668 |
| F _c ^{max} (h=7,A) | Linear | 1.02 [.] 10⁴+6.65 [.] h | 0.3997 | 0.1598 |
| | Hoerla | 1.16 [.] 10 ^{193.} 1.02 ^{h.} h ^{-62.22} | 0.8597 | 0.7391 |

Table 14. Regression function parameters with a correlation and goodness of fit values, R – Pearson correlation coefficient, R^2 – coefficient of the determination

Thus, it may be appropriate to test each convergent section type in parallel.

CONCLUSIONS

The output of the experimental study presented in this article is of simplified nature and concern a limited group of the extrusion die geometric features. Nonetheless, the results have confirmed the effect of the geometric features on the ultimate compression force value. This, in turn, is directly related to the extrudate density, considered the best end-product quality indicator.

Test results confirm that the type of shape and geometric parameters of channels influence the limit value of force required for extruding dry ice. Regression studies demonstrate that the height of the convergent section exhibits a nearly linear characteristic in almost all cases. In the authors' opinion, the nonlinear influence of the surface area suggests that these parameters should not be considered in future research. It can be noted, however, that in the case of research on multi-channel dies, no research results have been published so far regarding the influence of the geometric parameters of the shape of the input tapered cross-section on the value of the force boundary in the process. In this case, the studies to come shall be extended by investigation of the effect of the other geometrical features, including die cavity layout pattern, inlet diameter.

In addition, numerical analysis should be applied in further studies, to obtain more refined

information on multiple-cavity die parameters ensuring optimum efficiency of the dry ice production process. As a final effect, this will allow reduction of electricity consumption and decreasing the material sublimation rate. The primary overall benefit would be improved cost efficiency of dry ice application in cooling goods in transit.

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